

Design, Construction, and Test of a Miniature Parabolic Trough Solar Concentrator

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Abstract

This report describes the creation of a senior project. It recounts the steps taken from the initial design stages and choice of a parabolic trough solar concentrator, through the building process, and finally through the test stages of the project. The objective of the project was to test the capabilities of a Cal Poly trained engineer by performing a completely self guided process of designing, building, and testing a parabolic solar collection trough. This particular trough was designed to utilize the energy provided by the sun to heat water. The test results were compared with a mathematical model drafted during the design stage. Due to restrictions of cost and space, a miniature version of typical parabolic troughs used today was made. Because of the small scale of the project, actual use of the parabolic trough to heat water would not be feasible, although testing showed consistency with initial mathematical modeling.

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Background

The idea behind a concentrating solar collector is to minimize the heat losses associated with solar collection. In many instances it is desirable to deliver energy at higher temperatures than those possible with flat plate solar collectors. In this case, a parabolic “mirror” concentrates incident solar irradiation onto a much smaller receiver area, greatly decreasing heat loss and maximizing the available energy from the sun.

There are many different types of concentrating solar collectors in use today. Concentrators can be reflecting or refracting, cylindrical, spherical, parabolic, and they can be continuous or segmented. Receivers can be convex, flat, cylindrical, covered or uncovered. Because of the complexity and very wide scope of concentrators and concentrator designs, it is difficult to find developed general analyses of each specific type of concentrator. Therefore each solar concentrator design must be studied on a per case basis (Duffie, 324).

One important factor in the analysis of solar concentrators is the concentration ratio. The concentration ratio is defined as the ratio of the area of the aperture of the concentrator to the area of the receiver that is reflected upon by the concentrator. This is in essence the heart of a solar concentrator. Solar tracking is also necessary for efficient use of concentrating collectors. Without tracking the collector becomes almost useless except for a very short time period once a day. Large scale concentrators today use automated tracking systems that can track the sun on a biaxial path (Powell, 138-139). Due to cost restrictions and complexity, and the small scale of this project, manual turning of the concentrator was chosen as the preferred method of solar tracking.

The future of concentrating solar collectors will rely greatly on improved engineering, design, and materials. It is very important to maintain the quality of the optical systems of solar concentrators for long periods of time. This includes accounting for weather, dirt, and corrosion amongst other things (Eames, 748). The goal of this project is to design, build, model, and test a concentrating solar collector.

Design

The initial plan for a solar concentrator was to use a semi-spherical surface covered with many small sections of mirror to form a segmented, spherical concentrator. Referring to the optics section of a University Physics textbook it was found that the focal point of a spherical mirror would be located at a distance of half of the radius of the spherical section, directly above the vertex of the sphere. Quite some time was spent on trying to find a way to orient the small mirror sections at the proper angles about the inner surface of the sphere. The initial thought was to take the derivative of a circular equation to find the proper incline at different points along the sphere's inner surface. These inclines would then be rotated about the origin. This was a difficult problem considering the limited resources. A different approach was taken.

After conducting more research on solar energy and solar collection, the decision was made to attempt to build a parabolic trough solar concentrator. In a parabola all of the incoming rays from a light source (in this case the sun) are reflected back to the focal point of the parabola. If the said parabola is extended along an axis (becoming a trough) the solar rays are concentrated along a line through the focal point of the trough. The focal point of a parabola is located at $1/4a$, if the equation of the parabola is $y = ax^2$ (Young, 1157).

The parabolic trough selected fit the equation $y = .04167x^2$ from $x = -12$ inches to $x = 12$ inches. This equation was chosen to yield a focal point located at 6 inches above the vertex of the parabola, for ease of construction. Initial sketches and drawings are located in Appendix A.

A mathematical model was developed that would help determine the temperature of the water leaving the parabolic trough, knowing the temperature of the water entering the trough and the amount of insolation absorbed by the receiver. These calculations can be found in Appendix B.

The frame for the parabola was made out of plywood. It would be attached to a base which would allow for proper angling of the parabolic trough. The entire collector was small enough to allow for easy manual adjustment for solar tracking. The receiver chosen was a simple, half-inch copper pipe, painted black to absorb more incident radiation. Copper was chosen because of its high thermal conductivity and it is relatively inexpensive. The water source was planned as a reservoir located above the trough, with gravity assisted flow through the trough, to another reservoir used for collection of the heated water. These were going to be fifty gallon drums. But, based on the location of testing, a simple garden hose was used with a compression fitting to attach the hose to the copper pipe. This proved to be easier and more efficient, as well as significantly less expensive.

A piece of polished Aluminum was used for the reflective surface. A reasonably priced piece of aluminum was found at a website called OnlineMetals.com. This is a small quantity metals supplier owned by ThyssenKrupp. A 36 inch by 48 inch piece of 6061-T6 Aluminum, .020 inch thick was selected. With the design stage complete construction began.

Construction

The plan for construction consisted of the following. Based on the initial design sketches materials were bought at Home Depot. A 4 foot by 8 foot sheet of 23/32 inch outdoor plywood, four 8 foot 2 by 4's, a five foot long piece of $\frac{1}{2}$ inch copper pipe, black heat resistant spray paint, a can of spray on insulation, a carpenters square, an aluminum meter stick, measuring tape, and three boxes of different types of screws (1.25 inch wood screws, 2 inch wood screws, and 0.5 inch screws) were purchased.

The next step consisted of carefully drawing a grid of 1 inch squares on the plywood sheet, followed by a sub-grid of half inch squares. An origin and x and y axes were

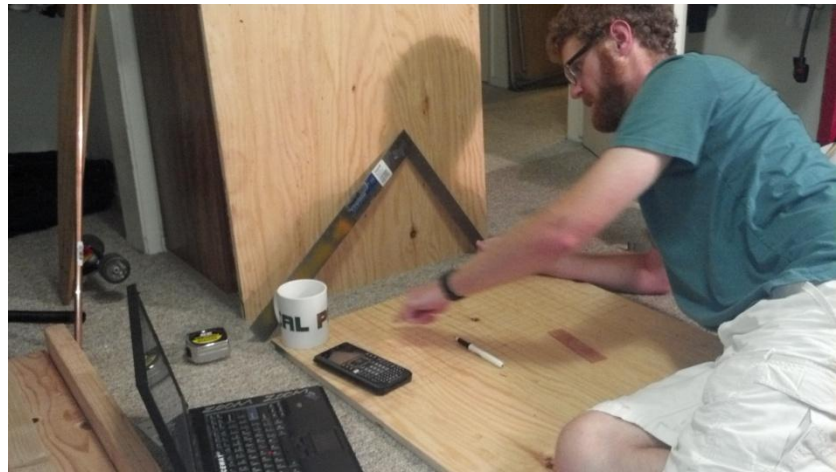


Figure 1: Forming the grid and the parabolic shape

chosen, and the points of the parabola were plotted (Figure 1). Next the 2 by 4's were measured and marked at the proper lengths. Drawings and dimensions can be found in Appendix A.

The dimensioned pieces of wood were brought to the woodshop at the Cal Poly Hangar. The table saw was used to cut the plywood grid into two pieces of the same size. The plan was to attach these two pieces, one atop the other, and cut them at the same time with a band saw. Consulting with two different shop techs at the hangar, it was concluded that the best method would be to cut one piece and use that as a template for the next piece. Also, the position of the band saw would make cutting the plywood difficult, so a jigsaw was used instead. Next the 2

by 4's were cut at the proper lengths using the miter saw. Finally, the two pieces of the parabolic frame plywood were attached and sanded to achieve the best possible parabolic curve. A half inch hole was drilled in each frame piece to serve as the point of rotation for angling of the trough, and the attachment to the base.

Next an attempt was made to attach two of the 2 by 4's to the outer edges of the parabolic frame pieces. One of the 2 by 4's had warped at some point, preventing the attachment of the two frame pieces. The warped board was discarded. It was then decided to use just one 2 by 4 to attach the two parabolic frame pieces. This proved to work out well. The 2 by 4 was attached to the rectangular base of the parabolic frames with three 2 inch screws on each side. It worked out to be a strong connection.

To build the base of the solar collector two partial triangles were cut from the remaining plywood. Then, with three of the pre-cut 2 by 4's the partial triangles were attached. Two small pieces of plywood were attached to each base plywood triangle and the top 2 by 4 for extra support. To these small rectangles of plywood two more pieces of plywood were attached, each with half inch holes drilled in them. These served as the points of attachment for the parabolic frame.

To attach the Aluminum sheet to the frame, it was cut into the proper size rectangle using the hangar's sheet metal cutter. Holes were drilled along the edges at increments of 2 inches to serve as places to screw the Aluminum to the plywood frame. The Aluminum sheet's polished side was protected by a plastic film. This film was left on the Aluminum sheet until immediately before testing. Next the Aluminum sheet was manually bent down into the parabolic frame and attached with the 0.5 inch screws. Some trouble was encountered here

because the Aluminum wasn't rigid enough to hold a consistent shape between the two frame pieces. It was too late to create another frame piece to help support the Aluminum, so it was decided that the Aluminum sheet could be supported in between the frames by attaching two 2 by 4's to the inside of the frame and securing them with screws. The problem was solved and the Aluminum parabola seemed to be nearly perfect.

The next items purchased included two pipe clamps, a compression fitting for the $\frac{1}{2}$ inch copper pipe, a straight coupling to attach the compression fitting to a standard garden hose, PTFE thread seal tape, and two $\frac{1}{2}$ inch bolts with nuts and washers. More plywood was measured and cut to attach to the sides of the parabolic frame. These pieces were for attaching the pipe clamps and running the copper pipe through the focal point of the parabola. Finally, two $\frac{1}{16}$ inch holes were drilled in the copper pipe at the inlet and outlet for placement of thermocouples. The copper pipe was sanded lightly and sprayed with flat black paint.



Figure 2: Completed Solar Concentrator

Having finished the construction, all that was left was putting it all together. The pieces were brought to the test site, and using the bolts, nuts, and washers the trough was attached to the base. The pipe clamps were loosened to attach the painted copper pipe to the trough. The threading of a garden hose was wrapped with PTFE tape and attached to the compression fitting on the end of the copper pipe. The solar concentrator was ready for testing.

Testing



Figure 3: Vernier Lab Cradle with TI-nSpire CX CAS

In order to test the solar concentrator testing equipment needed to be purchased. The first things needed were thermocouples. Online research led to the discovery of a company called Vernier Technology. Through this company a TI-nSpire Lab Cradle to connect with a TI-nSpire CX CAS calculator (Figure 3) were purchased. The lab cradle has five ports for different sensors. Two thermocouples to attach to the lab cradle were also purchased. The TI-nSpire CX calculator had software from Vernier pre-loaded, so all that

was necessary was to attach the calculator to the lab cradle and plug in the two thermocouples. An initial experiment showed that everything seemed to be working fine. Next a pyranometer (Figure 4) and an infrared thermometer (Figure 5) were purchased from the Amazon Online Marketplace, to measure the solar intensity and the temperatures of the surfaces of the solar concentrator, respectively.



Figure 4: Ambient Weather Pyranometer

Upon attempting to place the thermocouples into the holes drilled in the copper pipe it was found that the tip of one of the thermocouples was slightly larger than the other, so it did not fit into the hole drilled at the outlet of the copper pipe. For the first test the thermocouple

wire had to be shaped in a way that it would fit it into the end of the pipe and still be midstream in the water flow. The thermocouple was secured with tape and the previously drilled hole was sealed shut. The solar concentrator was positioned so that it was aimed directly at the sun. The two in-pipe midstream thermocouples were connected to the lab cradle/calculator set up and the Vernier software was opened.



Figure 5: HDE Infrared Thermometer

Data collection was started and the water valve was opened. Insolation values were measured at various times throughout the test. Attempts were made to measure the temperature of the surface of the pipe and the reflecting surface, but to no avail. The infrared thermometer may have been malfunctioning, as many reviews on Amazon had stated it would, or the temperatures were very widely varied on the surfaces of the concentrator.

The day of the first test was very hot, but very windy. This may have affected the results of the test. For the second test, a larger hole was drilled at the outlet of the copper pipe to accommodate the larger thermocouple tip. This way the outlet temperature could be measured more accurately. The second test



Figure 6: Prepping for testing

also yielded a much less windy day, which made a noticeable difference in the results.

Results

Testing went very smoothly. The initial test was highly influenced by the strong winds present and also the misplacement of the outlet thermocouple, not to mention an inconsistent inlet temperature. The cause of this inconsistency was unknown. Therefore the results will be deemed inconclusive (Figure 7). The second test, however, with minimal wind and a properly placed outlet thermocouple proved to be quite responsive (Figure 8). The initial temperature drops noticeable on both plots are attributable to solar heating of the water sitting in the hose prior to testing.

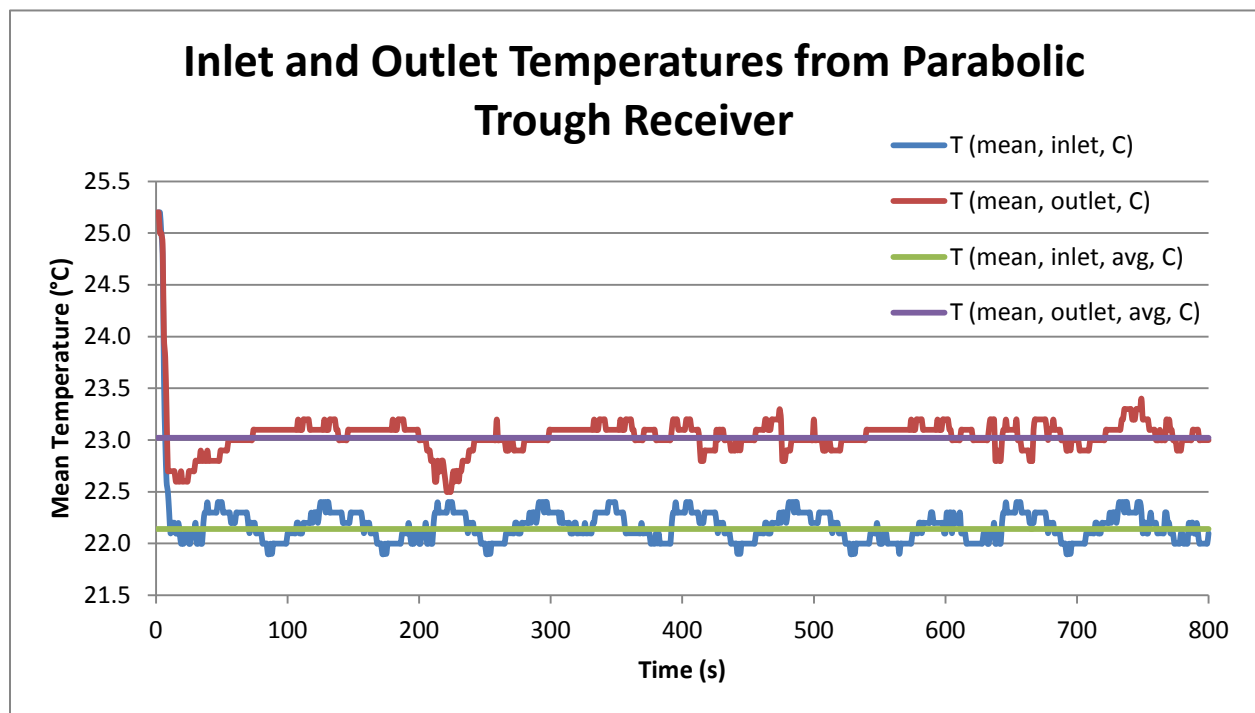


Figure 7: Test 1 of Solar Concentrator

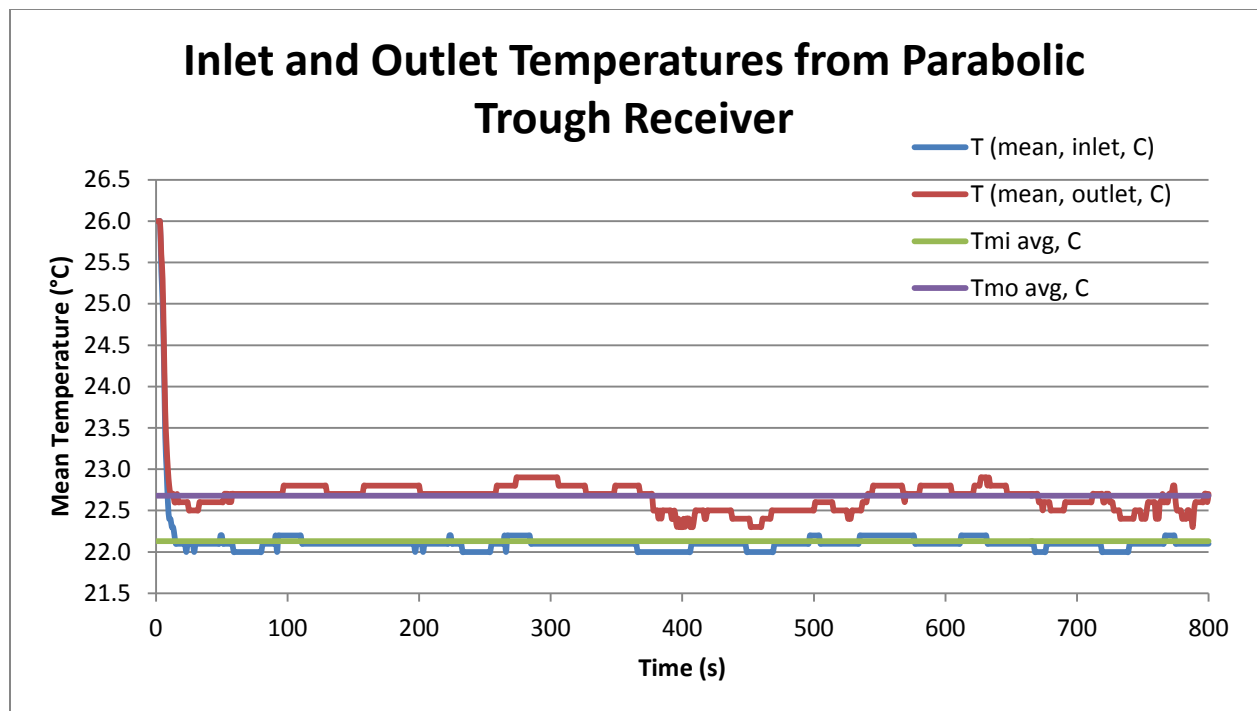


Figure 8: Test 2 of Solar Concentrator

The inlet and outlet temperatures were averaged across the total time of the test. The insolation measured at various times during testing was also averaged. The initial mathematical model was referred to and an expected outlet temperature was recalculated using the measured values. This model was then compared with the actual temperature recorded using the Vernier sensors. These calculations can be found in Appendix C. Data tables can be found in Appendix D.

Conclusion

The results achieved using this miniature parabolic trough solar concentrator did not quite match up to the results expected based on the initial mathematical modeling. The differences are shown in Appendix C. The mathematical model initially proposed considered perfect conditions and a perfect parabolic trough concentrator. Thermal losses can occur through heat transfer by conduction through the surface of the parabolic trough and into the frame. Heat loss due to convection can also be a major issue on the receiver pipe surface. Fluctuation in wind speed throughout testing likely caused more heat loss than expected. This could be combated by insulating the top surface of the receiver tube, or enclosing the trough using glass or Plexiglas with high transmittance. Better insulation of the back of the reflecting surface would also help with the heat lost due to conduction. The shape of the parabola may also not have been perfect due to issues during construction, such as imperfect jigsaw cutting, sanding of the frame, and mounting of the Aluminum. These could all affect the amount of incident radiation on the copper receiver tube and thus the amount of heat transferred to the concentrator fluid. These thermal losses likely reduced the amount of useful solar gain for my concentrator. Therefore the temperature gain received in testing was significantly lower than predicted with the mathematical model. Despite the thermal losses experienced the solar concentrator did heat the water fairly consistently throughout testing. The project was a general success through the processes of designing, building, and testing.

References

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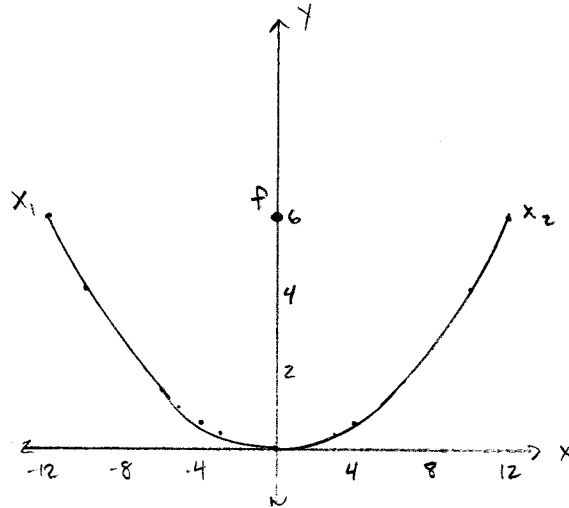
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Appendix A: Design Sketches

PARABOLIC TROUGH SOLAR COLLECTOR

EQUATION OF PARABOLA: $y = .041667x^2$ (unit = 1 in)

FOCAL POINT AT $\frac{1}{4a} = \frac{1}{4(.041667)} = 6$ in ABOVE ORIGIN



LENGTH OF PARABOLA FROM x_1 TO x_2 , "S"

LET $t = 2f$ and $q = \sqrt{t^2 + p^2}$, p IS DISTANCE FROM y -AXIS TO POINT x

$$S = \left(\frac{pq}{t} + t \ln \left(\frac{p+q}{t} \right) \right)$$

$$p = 12$$

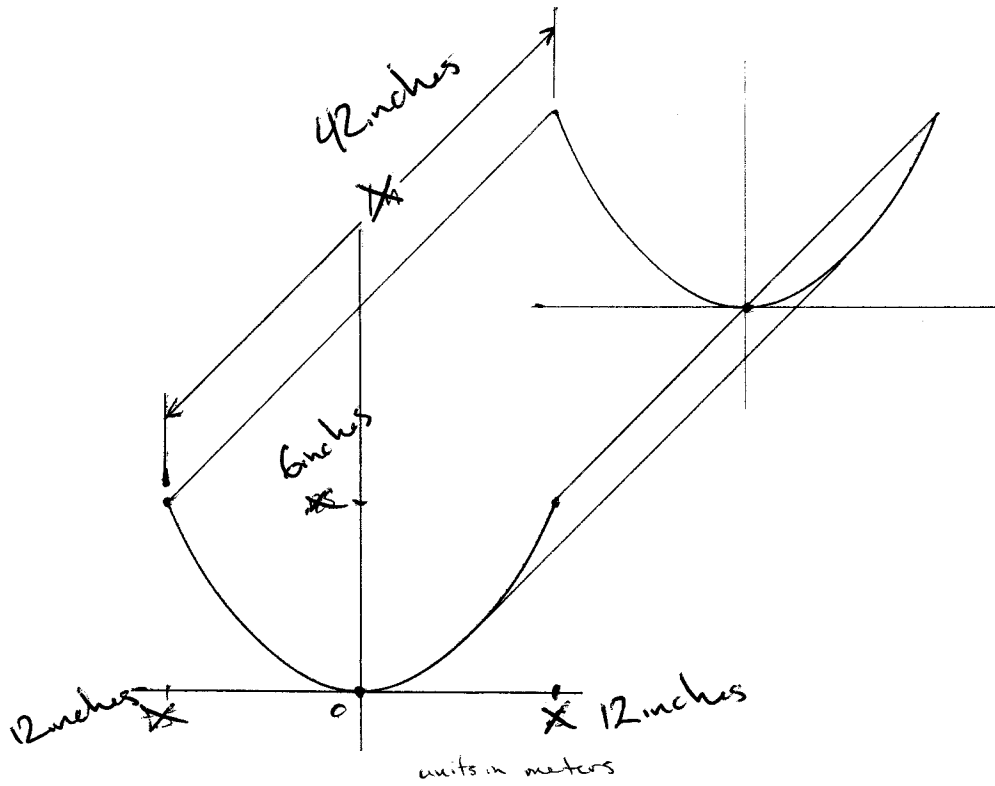
$$t = 2(6) = 12$$

$$q = \sqrt{2 \cdot 12^2} = 1.697 \times 10'$$

$$S = \left(\frac{12 \cdot 16.97}{12} + 12 \ln \left(\frac{12 + 16.97}{12} \right) \right)$$

$$S = 27.55 \text{ in}$$

TOTAL LENGTH NEEDED FOR SHEET METAL = 27.55 in



The trough will be ~~42~~ in length as shown (not to scale)

42 inches

↓

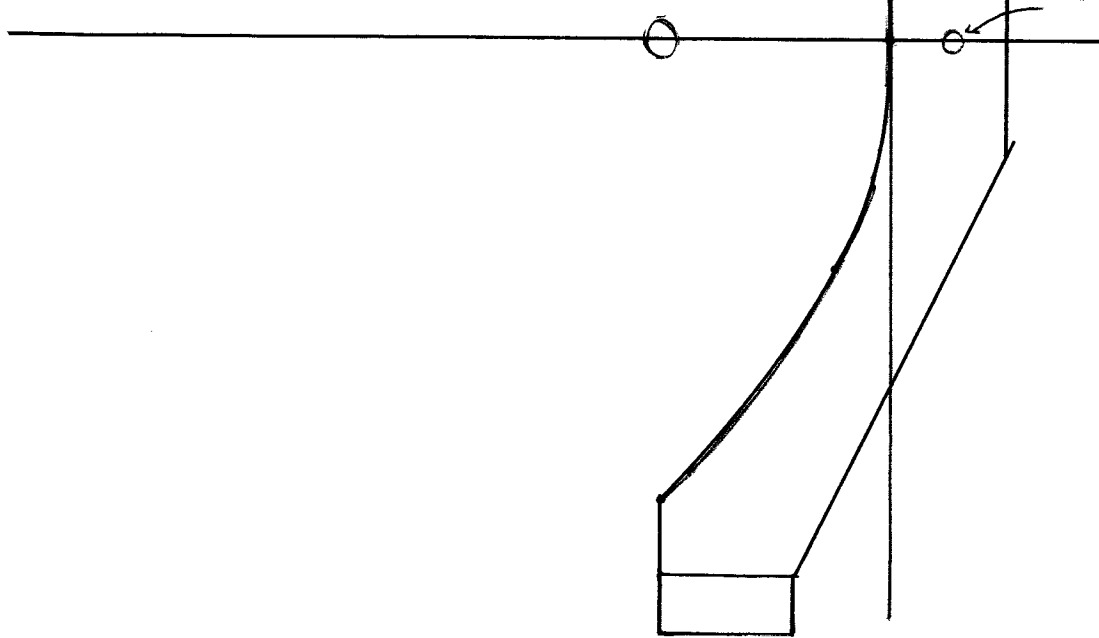
Changed to 40 inches

1 square = 1 inch
 3/4 inch plywood

2 x 2 x 4 supports
 w/ 2 x 2 in
 wood screws

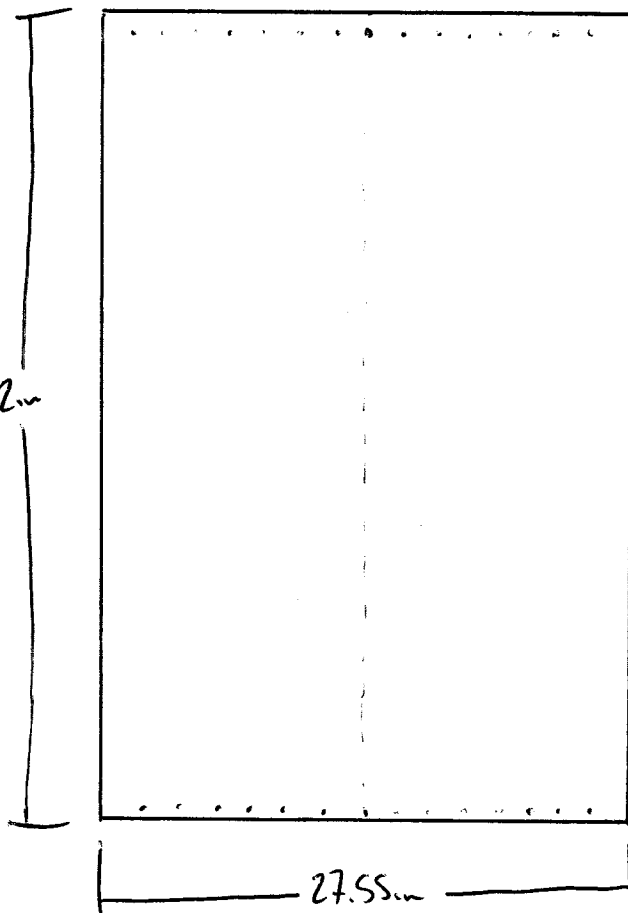
42 inches

Hole Drilled 1/2 inch



Changed to
40 inches

42 in



Drill holes every 2 inches
from ~~the~~ center line
hole

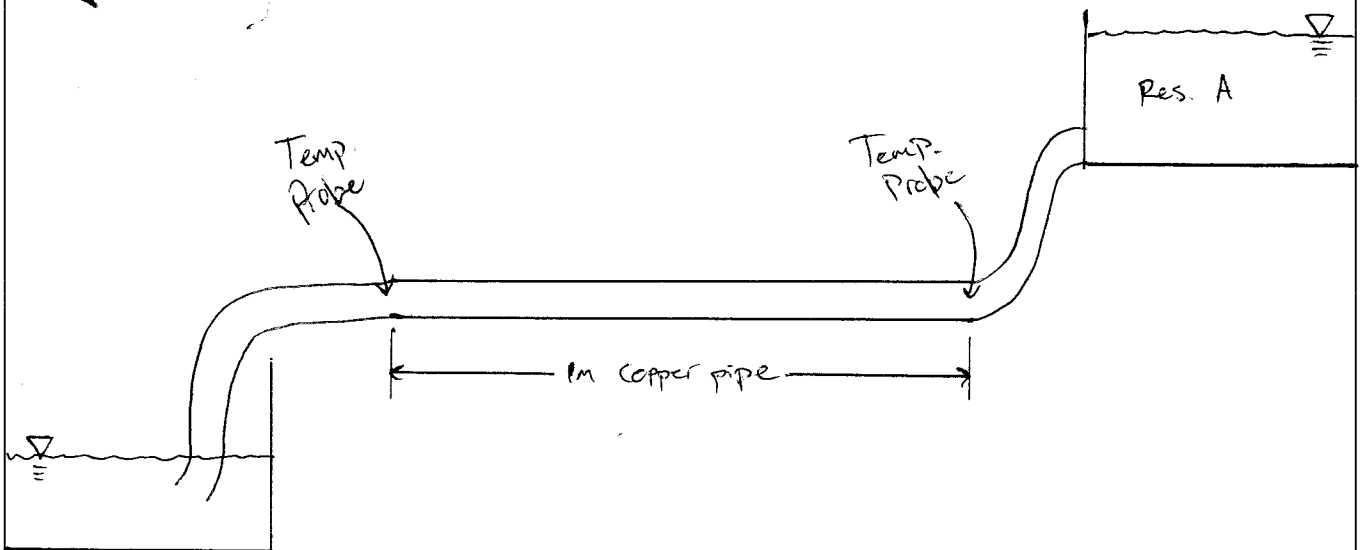
Reflective Surface

6061 T6 Aluminum

Collector / receiver

I plan to use a $\frac{1}{2}$ inch copper pipe as my receiver. The pipe will be sprayed with black heat resistant paint to maximize the energy absorbed by the copper pipe. Copper will be used for its high thermal conductivity. The pipe will be placed at the focal point of the parabolic trough, spanning the entire trough. Water will be run through the copper pipe while the temperature of the water will be measured at the inlet and outlet of the pipe.

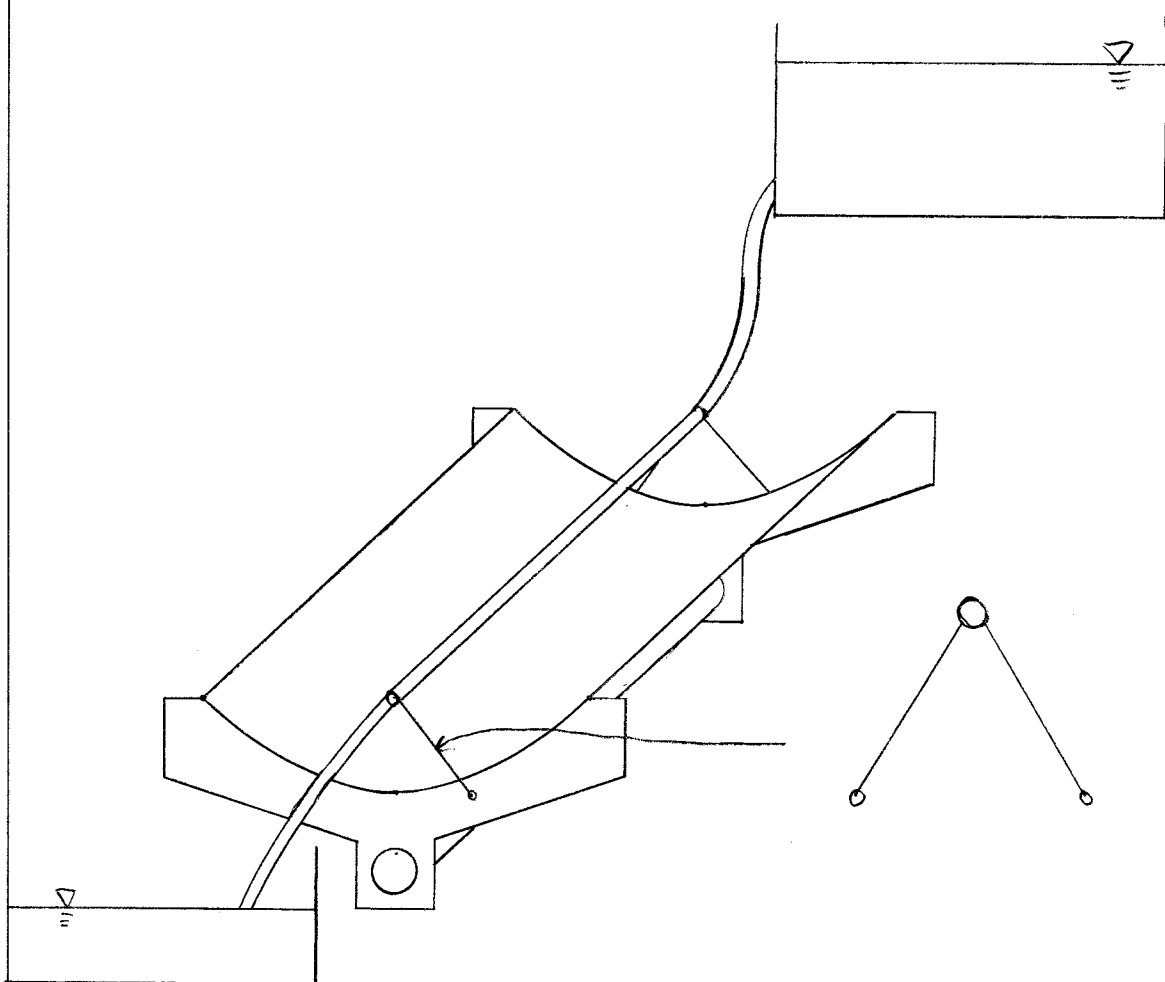
A simple siphon will be used to provide the water flow, with an upper reservoir large enough to allow ample time for testing.



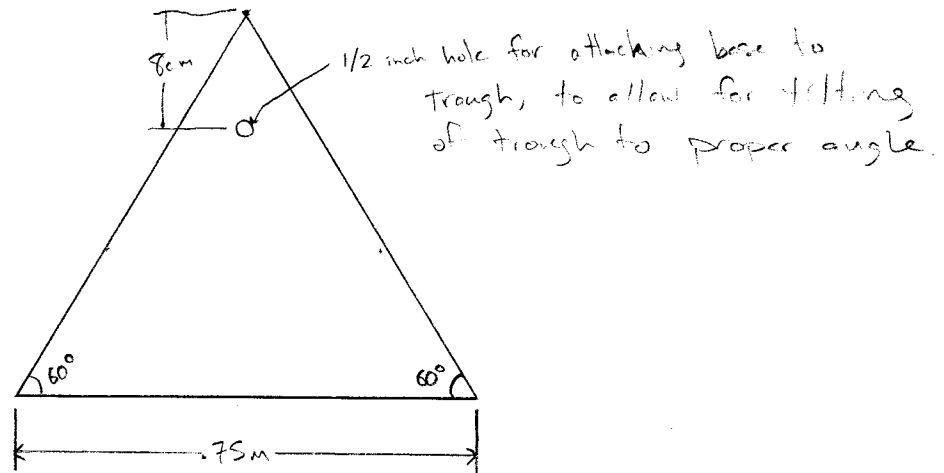
The pipes connecting the copper pipes to the reservoirs will be flexible, thermally resistant PEX tubing, also $\frac{1}{2}$ inch.

My first idea for the upper reservoir A is to use a large camp cooler to attempt to keep the reservoir temperature at a constant level, hopefully this will be large enough for my purposes.

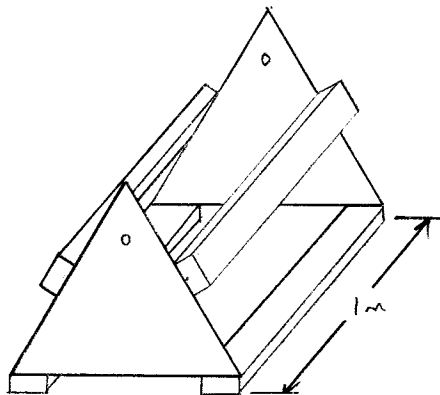
The lower reservoir is not as important, just a tub to collect the water for retesting purposes...



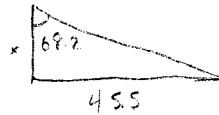
Base - Simple triangular plywood pieces (2)
 Held together by 2 by 1/2 in base of triangle



Base

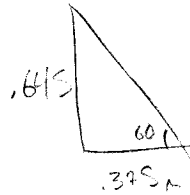
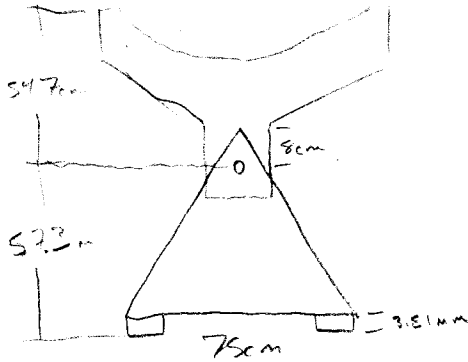


Appendix B: Mathematical Modeling



$$\tan 68.2 = \frac{45.5}{x}$$

$$x = 18.2 \text{ cm}$$

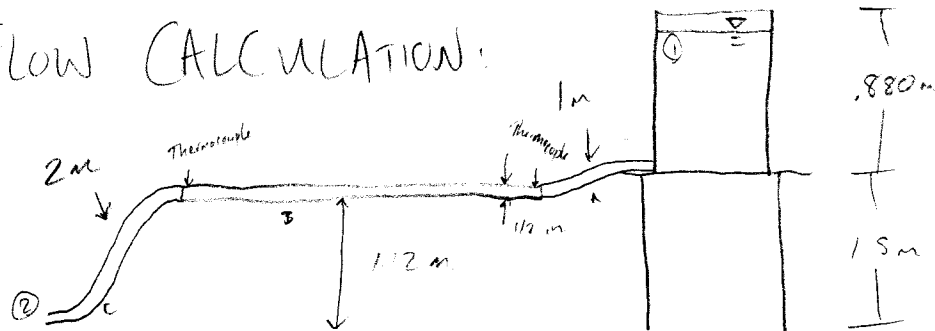


$$\tan 60 = \frac{0.615}{0.375}$$

Overall height = 1.12 m

↳ Table for SS gal drum at 1.5 m

Flow CALCULATION:



Energy Equation

$$h_p - h_T = h_{out} - h_{in} + h_{loss}$$

$$h_p = h_{out} + h_{loss}$$

$$\frac{P_1}{\rho} + z_1 + \frac{V_1^2}{2g} = \frac{P_2}{\rho} + z_2 + \frac{V_2^2}{2g} + h_{loss} + h_{in}$$

$$z_1 = z_2 + h_{in} + h_{loss}$$

$$z_1 = h_{in} + h_{loss}$$

$$z_1 = f \frac{L}{D} \frac{V^2}{2g} + K \frac{V^2}{2g}$$

$$z_1 = f_A \frac{L_A}{D_A} \frac{V^2}{2g} + f_B \frac{L_B}{D_B} \frac{V^2}{2g} + f_C \frac{L_C}{D_C} \frac{V^2}{2g} + K \frac{V^2}{2g}$$

k_s for Vex Tubes: .007 mm, $D = 12.7$ mm $\frac{k_s}{D} = .0005$

k_s for copper pipe: .0015 mm, $D = 12.7$ mm $\frac{k_s}{D} = .0001$

$$Z_1 = \frac{V^2}{2g} \left(f_A \left(\frac{L_A}{D_A} + \frac{L_C}{D_C} \right) + f_B \frac{L_B}{D_B} + K \right)$$

$K = 2(.08) + 1 + .8 + 2$
 unions exit entrance valve
 $= 5.4$

guess $f_A = .02$, $f_B = .018$

$$2.38 \text{ m} = \frac{V^2}{2(9.81 \frac{\text{m}}{\text{s}^2})} \left[.02 \left(\frac{1 \text{ m}}{.0127 \text{ m}} + \frac{2 \text{ m}}{.0127 \text{ m}} \right) + .018 \left(\frac{1 \text{ m}}{.0127 \text{ m}} \right) + 5.4 \right]$$

$$V^2 = 1.4 \frac{\text{m}^2}{\text{s}^2}$$

$$V = 1.183 \text{ m/s}$$

$\rightarrow Re = \frac{VD}{\nu} = \frac{(1.183 \text{ m/s})(.0127 \text{ m})}{1.004 \times 10^{-6} \text{ m}^2/\text{s}} = 1.5 \times 10^4$
 assume 20°C assume temp change in ν due to effect of V are negligible

\rightarrow Find new friction factors on moody chart
 $f_A = .029$ $f_B = .029$

$$Z_1 = \frac{V^2}{2g} f \left(\frac{L_A}{D_A} + \frac{L_C}{D_C} + \frac{L_B}{D_B} + K \right)$$

$$2.38 \text{ m} = \frac{V^2}{2(9.81)} (.029) \left(\frac{1 \text{ m}}{.0127 \text{ m}} + \frac{2 \text{ m}}{.0127 \text{ m}} + \frac{1 \text{ m}}{.0127 \text{ m}} + 5.4 \right)$$

$$V = 1.06 \text{ m/s}$$

$\rightarrow Re = \frac{VD}{\nu} = \frac{(1.06 \text{ m/s})(.0127 \text{ m})}{1.004 \times 10^{-6} \text{ m}^2/\text{s}} = 1.34 \times 10^4$

f remains at .029

V in pipes: 1.06 m/s

$Q = AV = \frac{\pi}{4} (.0127 \text{ m})^2 (1.06 \text{ m/s}) = 1.3 \times 10^{-4} \text{ m}^3/\text{s} \times \frac{1 \text{ gal}}{.00378 \text{ m}^3} = 3.55 \times 10^{-2} \frac{\text{gal}}{\text{s}}$

Revised Mathematical Model
(After purchasing & reading solar engineering of Thermal Processes ...)

Area Concentration Ratio:

APERTURE AREA : Receiver Area

$$C = \frac{A_a}{A_r} = \frac{(42 \text{ m}) \times (24 \text{ m})}{\left[\frac{\pi \times \left(\frac{5}{8}\right)^2}{4} \right] (42 \text{ m})}$$

$$C = 78.2$$

Assuming Insolation of 800 W/m^2

$$q_r'' = C \times I \times \rho \times K$$

I = Insolation
 ρ = reflectivity of concentrator
 K = Correction factor ex. 7 for this

$$= 78.2 \times 800 \times .8 \times .7$$

$$= 35,000 \text{ W/m}^2$$

Assume Constant heating condition on tube surface,

Concentrator Fluid - H_2O

$$\rho = 1000 \text{ kg/m}^3$$

$$k = .58 \text{ W/m}\cdot\text{K}$$

$$C_p = 4210 \text{ J/kg}\cdot\text{K}$$

$$\mu = 1652 \times 10^{-6} \text{ N}\cdot\text{s/m}^2$$

Tube Diameter = 12.7 mm ($\frac{1}{2} \text{ inch}$)

Mass flow rate: From Flow CALCULATION

$$\dot{M} = 3.55 \times 10^{-2} \frac{\text{gal}}{\text{s}} \times \frac{3.7854 \text{ kg}}{1 \text{ gal}} = .1344 \text{ kg/s}$$

Assume: Steady State, Incompressible,
Constant Properties, Thin tube wall.

From: INTRO TO HEAT TRANSFER

$$A_s = \pi D L = \frac{\dot{m} C_p (T_{m,o} - T_{m,i})}{q_s''}$$

$$\frac{\pi D L q_s''}{\dot{m} C_p} + T_{m,i} = T_{m,o}$$

$$D = 12.7 \text{ mm} \checkmark$$

$$L = 42 \text{ in} \rightarrow 1.066 \text{ m} \checkmark$$

$$q_s'' = 35,000 \text{ W/m}^2 \checkmark$$

$$\dot{m} = 0.1344 \text{ kg/s} \checkmark$$

$$C_p = 4210 \text{ J/kg K}$$

$$\rightarrow \text{Assume } T_{m,i} = 72^\circ \text{F} = 22.22^\circ \text{C}$$

$$\frac{\pi (.0127 \text{ m})(1.066 \text{ m})(35,000 \text{ W/m}^2)}{(0.1344 \text{ kg/s})(4210 \text{ J/kg K})} + 22.22^\circ \text{C} = T_{m,o}$$

$$2.63 + 22.22 = T_{m,o}$$

$$T_{m,o} = 24.85^\circ \text{C}$$

Appendix C: Comparing Test Results to Mathematical Modeling

After building and Testing:

Area Concentration Ratio:

* During Construction length of trough changed to 40 inches

$$C = \frac{A_a}{A_r} = \frac{(40 \text{ in})(24 \text{ in})}{(\pi/4 \times (\frac{5}{8})^2) \times 40 \text{ in}}$$

$$C = 78.2$$

From ch 7 IN Solar engineering of thermal processes:

$$q_s'' = C \times I \times \rho \times K$$

I: INSOLATION

ρ : reflectivity of collector - Assume .8

K: Correction value - Assume .7 for average collectors

$$q_s'' = 78.2 \times 1046 \frac{\text{W}}{\text{m}^2} \times .8 \times .7$$

$$q_s'' = \underline{\underline{45,800 \text{ W/m}^2}}$$

Still assuming Constant heating condition on tube surface, same fluid, same tube Diameter

$$\text{Measured Mass flow rate: } 3.86 \frac{\text{oz}}{\text{s}} \times \frac{.0078125 \text{ gal}}{\text{oz}}$$

$$= .03016 \frac{\text{gal}}{\text{s}} \times \frac{3.7854 \text{ kg}}{\text{gal}}$$

$$= .114 \text{ kg/s}$$

Assume Steady State, Incompressible, constant properties, thin tube wall

$$A_s = \pi D L = \frac{\dot{m} C_p (T_{m,o} - T_{m,i})}{q_s''} \quad \left. \vphantom{\frac{\dot{m} C_p (T_{m,o} - T_{m,i})}{q_s''}} \right\} \text{From Intro to heat transfer}$$

$$D = 12.7 \text{ mm}$$

$$L = 40 \text{ m} \rightarrow 1.016 \text{ m}$$

$$q_s'' = 45,800 \text{ W/m}^2$$

$$\dot{m} = .114 \text{ kg/s}$$

$$C_p = 4210 \text{ J/kg} \cdot \text{K}$$

$$\rightarrow \text{measured } T_{m,i} = 22.13^\circ\text{C}$$

$$T_{m,o} = 22.68^\circ\text{C}$$

$$\frac{\pi D L q_s''}{\dot{m} C_p} + T_{m,i} = T_{m,o}$$

$$\frac{\pi (12.7 \times 10^{-3} \text{ m}) (1.016 \text{ m}) (45,800 \text{ W/m}^2)}{(.114 \text{ kg/s}) (4210 \text{ J/kg} \cdot \text{K})} + 22.13^\circ\text{C}$$

$$3.87 + 22.13^\circ\text{C} = T_{m,o}$$

$$26^\circ\text{C} = T_{m,o}$$

$$\rightarrow \text{Actual } T_{m,o} = 22.68^\circ\text{C}$$

Calculated $T_{m,o}$ is higher than $T_{m,o}$ measured.

Appendix D: Data Tables

Test 1:

Time (s)	T (mean, inlet, C)	T (mean, outlet, C)
1	25.2	25.2
2	25.2	25.2
3	25.2	25.0
4	25.0	25.0
5	24.8	24.9
6	23.6	24.0
7	23.0	23.8
8	22.6	23.3
9	22.5	22.7
10	22.3	22.7
11	22.1	22.7
12	22.2	22.7
13	22.2	22.7
14	22.2	22.7
15	22.1	22.6
16	22.1	22.6
17	22.2	22.6
18	22.1	22.6
19	22.0	22.7
20	22.0	22.6
21	22.0	22.6
22	22.1	22.6
23	22.1	22.6
24	22.1	22.6
25	22.0	22.7
26	22.0	22.7
27	22.1	22.7
28	22.1	22.7
29	22.1	22.7
30	22.2	22.8
31	22.0	22.8
32	22.0	22.8
33	22.1	22.8
34	22.0	22.9
35	22.0	22.8
36	22.2	22.8

37	22.3	22.8
38	22.3	22.8
39	22.4	22.9
40	22.3	22.8
41	22.3	22.8
42	22.3	22.8
43	22.3	22.8
44	22.3	22.8
45	22.3	22.8
46	22.3	22.8
47	22.4	22.8
48	22.4	22.8
49	22.4	22.9
50	22.4	22.9
51	22.3	22.9
52	22.3	22.9
53	22.3	22.9
54	22.3	22.9
55	22.3	23.0
56	22.3	23.0
57	22.2	23.0
58	22.2	23.0
59	22.2	23.0
60	22.2	23.0
61	22.3	23.0
62	22.3	23.0
63	22.3	23.0
64	22.3	23.0
65	22.3	23.0
66	22.3	23.0
67	22.3	23.0
68	22.3	23.0
69	22.3	23.0
70	22.2	23.0
71	22.1	23.0
72	22.2	23.0
73	22.2	23.0
74	22.2	23.1
75	22.2	23.1
76	22.1	23.1
77	22.1	23.1
78	22.1	23.1

79	22.1	23.1
80	22.1	23.1
81	22.0	23.1
82	22.0	23.1
83	22.0	23.1
84	22.0	23.1
85	21.9	23.1
86	21.9	23.1
87	22.0	23.1
88	21.9	23.1
89	22.0	23.1
90	22.0	23.1
91	22.0	23.1
92	22.0	23.1
93	22.0	23.1
94	22.0	23.1
95	22.0	23.1
96	22.0	23.1
97	22.0	23.1
98	22.0	23.1
99	22.0	23.1
100	22.1	23.1
101	22.1	23.1
102	22.1	23.1
103	22.1	23.1
104	22.1	23.1
105	22.1	23.1
106	22.1	23.1
107	22.2	23.1
108	22.2	23.2
109	22.1	23.1
110	22.1	23.1
111	22.1	23.1
112	22.2	23.2
113	22.2	23.2
114	22.2	23.2
115	22.2	23.2
116	22.2	23.2
117	22.2	23.1
118	22.3	23.1
119	22.2	23.1
120	22.3	23.1

121	22.3	23.1
122	22.3	23.1
123	22.2	23.1
124	22.3	23.1
125	22.4	23.1
126	22.4	23.1
127	22.4	23.1
128	22.4	23.2
129	22.3	23.1
130	22.4	23.1
131	22.4	23.1
132	22.3	23.2
133	22.3	23.2
134	22.3	23.2
135	22.3	23.2
136	22.3	23.2
137	22.3	23.1
138	22.3	23.1
139	22.3	23.0
140	22.3	23.0
141	22.3	23.0
142	22.2	23.0
143	22.2	23.0
144	22.2	23.0
145	22.2	23.0
146	22.2	23.1
147	22.2	23.1
148	22.3	23.1
149	22.3	23.1
150	22.3	23.1
151	22.3	23.1
152	22.3	23.1
153	22.3	23.1
154	22.3	23.1
155	22.3	23.1
156	22.3	23.1
157	22.2	23.1
158	22.1	23.1
159	22.2	23.1
160	22.2	23.1
161	22.2	23.1
162	22.2	23.1

163	22.1	23.1
164	22.1	23.1
165	22.1	23.1
166	22.1	23.1
167	22.1	23.1
168	22.0	23.1
169	22.0	23.1
170	22.0	23.1
171	22.0	23.1
172	21.9	23.1
173	21.9	23.1
174	22.0	23.1
175	21.9	23.1
176	22.0	23.1
177	22.0	23.1
178	22.0	23.1
179	22.0	23.1
180	22.0	23.2
181	22.0	23.1
182	22.0	23.1
183	22.0	23.1
184	22.0	23.2
185	22.0	23.2
186	22.0	23.2
187	22.1	23.2
188	22.1	23.2
189	22.1	23.1
190	22.1	23.1
191	22.1	23.1
192	22.2	23.1
193	22.1	23.1
194	22.0	23.1
195	22.0	23.1
196	22.0	23.1
197	22.1	23.1
198	22.1	23.1
199	22.1	23.1
200	22.0	23.0
201	22.0	23.0
202	22.1	23.0
203	22.1	23.0
204	22.1	23.0

205	22.2	23.0
206	22.0	22.9
207	22.0	22.9
208	22.1	22.9
209	22.0	22.8
210	22.0	22.8
211	22.2	22.8
212	22.3	22.6
213	22.3	22.6
214	22.4	22.8
215	22.3	22.7
216	22.3	22.7
217	22.3	22.8
218	22.3	22.8
219	22.3	22.7
220	22.3	22.6
221	22.3	22.5
222	22.4	22.5
223	22.4	22.5
224	22.4	22.5
225	22.4	22.6
226	22.3	22.7
227	22.3	22.6
228	22.3	22.7
229	22.3	22.7
230	22.3	22.6
231	22.3	22.7
232	22.3	22.7
233	22.3	22.8
234	22.3	22.8
235	22.3	22.8
236	22.2	22.8
237	22.1	22.8
238	22.2	22.9
239	22.2	22.9
240	22.2	22.9
241	22.2	22.9
242	22.1	23.0
243	22.1	23.0
244	22.1	23.0
245	22.1	23.0
246	22.1	23.0

247	22.0	23.0
248	22.0	23.0
249	22.0	23.0
250	22.0	23.0
251	21.9	23.0
252	21.9	23.0
253	22.0	23.0
254	21.9	23.0
255	22.0	23.0
256	22.0	23.0
257	22.0	23.0
258	22.0	23.0
259	22.0	23.2
260	22.0	23.0
261	22.0	23.0
262	22.0	23.0
263	22.0	23.0
264	22.0	23.0
265	22.0	23.0
266	22.1	22.9
267	22.1	22.9
268	22.1	22.9
269	22.1	23.0
270	22.1	23.0
271	22.1	23.0
272	22.1	22.9
273	22.2	22.9
274	22.2	22.9
275	22.1	22.9
276	22.1	22.9
277	22.1	22.9
278	22.2	22.9
279	22.2	22.9
280	22.2	23.0
281	22.2	23.0
282	22.2	23.0
283	22.2	23.0
284	22.3	23.0
285	22.2	23.0
286	22.3	23.0
287	22.3	23.0
288	22.3	23.0

289	22.2	23.0
290	22.3	23.0
291	22.4	23.0
292	22.4	23.0
293	22.4	23.0
294	22.4	23.0
295	22.3	23.0
296	22.4	23.0
297	22.4	23.0
298	22.3	23.0
299	22.3	23.1
300	22.3	23.1
301	22.3	23.1
302	22.3	23.1
303	22.3	23.1
304	22.3	23.1
305	22.3	23.1
306	22.3	23.1
307	22.2	23.1
308	22.3	23.1
309	22.2	23.1
310	22.2	23.1
311	22.2	23.1
312	22.2	23.1
313	22.2	23.1
314	22.2	23.1
315	22.1	23.1
316	22.1	23.1
317	22.1	23.1
318	22.2	23.1
319	22.2	23.1
320	22.1	23.1
321	22.1	23.1
322	22.1	23.1
323	22.1	23.1
324	22.1	23.1
325	22.1	23.1
326	22.2	23.1
327	22.1	23.1
328	22.1	23.1
329	22.1	23.1
330	22.1	23.1

331	22.1	23.1
332	22.2	23.1
333	22.2	23.2
334	22.3	23.1
335	22.3	23.1
336	22.3	23.1
337	22.3	23.2
338	22.3	23.2
339	22.3	23.2
340	22.3	23.2
341	22.3	23.2
342	22.3	23.1
343	22.3	23.1
344	22.4	23.1
345	22.4	23.1
346	22.4	23.1
347	22.4	23.1
348	22.4	23.1
349	22.4	23.1
350	22.3	23.1
351	22.3	23.1
352	22.3	23.1
353	22.3	23.2
354	22.3	23.1
355	22.3	23.1
356	22.2	23.1
357	22.1	23.2
358	22.1	23.2
359	22.1	23.2
360	22.1	23.2
361	22.1	23.2
362	22.1	23.1
363	22.1	23.1
364	22.1	23.0
365	22.1	23.0
366	22.1	23.0
367	22.1	23.0
368	22.1	23.0
369	22.2	23.0
370	22.1	23.0
371	22.1	23.1
372	22.1	23.1

373	22.1	23.1
374	22.1	23.1
375	22.1	23.1
376	22.0	23.1
377	22.0	23.1
378	22.0	23.1
379	22.0	23.1
380	22.1	23.0
381	22.0	23.0
382	22.1	23.0
383	22.0	23.0
384	22.0	23.0
385	22.0	23.0
386	22.0	23.0
387	22.0	23.0
388	22.0	23.0
389	22.0	23.0
390	22.0	23.0
391	22.0	23.0
392	22.2	23.1
393	22.3	23.2
394	22.3	23.2
395	22.4	23.2
396	22.3	23.2
397	22.3	23.1
398	22.3	23.1
399	22.3	23.1
400	22.3	23.1
401	22.3	23.1
402	22.3	23.1
403	22.4	23.2
404	22.4	23.2
405	22.4	23.2
406	22.4	23.2
407	22.3	23.1
408	22.3	23.1
409	22.3	23.1
410	22.3	23.1
411	22.3	23.1
412	22.3	23.1
413	22.2	23.0
414	22.2	22.8

415	22.2	22.8
416	22.2	22.8
417	22.3	22.9
418	22.3	22.9
419	22.3	22.9
420	22.3	22.9
421	22.3	22.9
422	22.3	22.9
423	22.3	22.9
424	22.3	22.9
425	22.3	22.9
426	22.2	23.1
427	22.1	23.0
428	22.2	23.1
429	22.2	23.1
430	22.2	23.1
431	22.2	23.1
432	22.1	23.0
433	22.1	23.0
434	22.1	23.0
435	22.1	23.0
436	22.1	23.0
437	22.0	22.9
438	22.0	22.9
439	22.0	22.9
440	22.0	22.9
441	22.0	23.0
442	21.9	22.9
443	22.0	23.0
444	21.9	22.9
445	22.0	23.0
446	22.0	23.0
447	22.0	23.0
448	22.0	23.0
449	22.0	23.0
450	22.0	23.0
451	22.0	23.0
452	22.0	23.0
453	22.0	23.0
454	22.0	23.0
455	22.0	23.0
456	22.1	23.1

457	22.1	23.1
458	22.1	23.1
459	22.1	23.1
460	22.1	23.1
461	22.1	23.1
462	22.1	23.1
463	22.2	23.2
464	22.2	23.2
465	22.1	23.1
466	22.1	23.1
467	22.1	23.1
468	22.2	23.2
469	22.2	23.2
470	22.2	23.2
471	22.2	23.2
472	22.2	23.2
473	22.2	23.2
474	22.3	23.3
475	22.2	23.2
476	22.3	22.8
477	22.3	22.8
478	22.3	22.8
479	22.2	22.9
480	22.3	22.9
481	22.4	22.9
482	22.4	22.9
483	22.4	23.0
484	22.4	23.0
485	22.3	23.0
486	22.4	23.0
487	22.4	23.0
488	22.3	23.0
489	22.3	23.0
490	22.3	23.0
491	22.3	23.0
492	22.3	23.0
493	22.3	23.0
494	22.3	23.0
495	22.3	23.0
496	22.3	23.0
497	22.3	23.0
498	22.2	23.0

499	22.2	23.0
500	22.2	23.2
501	22.2	23.0
502	22.2	23.0
503	22.2	23.0
504	22.3	23.0
505	22.3	23.0
506	22.3	23.0
507	22.3	22.9
508	22.3	22.9
509	22.3	22.9
510	22.3	23.0
511	22.3	23.0
512	22.3	23.0
513	22.2	22.9
514	22.1	22.9
515	22.2	22.9
516	22.2	22.9
517	22.2	22.9
518	22.2	22.9
519	22.1	22.9
520	22.1	22.9
521	22.1	23.0
522	22.1	23.0
523	22.1	23.0
524	22.0	23.0
525	22.0	23.0
526	22.0	23.0
527	22.0	23.0
528	21.9	23.0
529	21.9	23.0
530	22.0	23.0
531	21.9	23.0
532	22.0	23.0
533	22.0	23.0
534	22.0	23.0
535	22.0	23.0
536	22.0	23.0
537	22.0	23.0
538	22.0	23.0
539	22.0	23.0
540	22.0	23.1

541	22.0	23.1
542	22.0	23.1
543	22.1	23.1
544	22.1	23.1
545	22.1	23.1
546	22.1	23.1
547	22.1	23.1
548	22.2	23.1
549	22.1	23.1
550	22.0	23.1
551	22.0	23.1
552	22.0	23.1
553	22.1	23.1
554	22.1	23.1
555	22.1	23.1
556	22.0	23.1
557	22.0	23.1
558	22.0	23.1
559	22.0	23.1
560	22.0	23.1
561	22.0	23.1
562	22.0	23.1
563	22.0	23.1
564	22.0	23.1
565	21.9	23.1
566	22.0	23.1
567	22.0	23.1
568	22.0	23.1
569	22.0	23.1
570	22.0	23.1
571	22.0	23.1
572	22.0	23.1
573	22.0	23.1
574	22.0	23.2
575	22.1	23.1
576	22.1	23.1
577	22.1	23.1
578	22.1	23.2
579	22.1	23.2
580	22.2	23.2
581	22.2	23.2
582	22.1	23.2

583	22.2	23.1
584	22.2	23.1
585	22.2	23.1
586	22.2	23.1
587	22.2	23.1
588	22.2	23.1
589	22.3	23.1
590	22.3	23.1
591	22.2	23.1
592	22.2	23.1
593	22.2	23.1
594	22.2	23.2
595	22.2	23.1
596	22.2	23.1
597	22.2	23.1
598	22.2	23.2
599	22.1	23.2
600	22.1	23.2
601	22.3	23.2
602	22.2	23.2
603	22.3	23.1
604	22.3	23.1
605	22.1	23.0
606	22.3	23.0
607	22.2	23.0
608	22.2	23.0
609	22.2	23.0
610	22.2	23.0
611	22.3	23.0
612	22.1	23.1
613	22.1	23.1
614	22.1	23.1
615	22.1	23.1
616	22.0	23.1
617	22.0	23.1
618	22.0	23.1
619	22.0	23.1
620	22.0	23.1
621	22.0	23.0
622	22.0	23.0
623	22.0	23.0
624	22.0	23.0

625	22.0	23.0
626	22.0	23.0
627	22.0	23.0
628	22.0	23.0
629	22.1	23.0
630	22.1	23.0
631	22.1	23.0
632	22.0	23.0
633	22.0	23.1
634	22.1	23.2
635	22.1	23.2
636	22.1	23.2
637	22.2	23.2
638	22.0	22.8
639	22.0	22.8
640	22.1	22.9
641	22.0	22.8
642	22.0	22.8
643	22.2	23.0
644	22.3	23.1
645	22.3	23.1
646	22.4	23.2
647	22.3	23.1
648	22.3	23.1
649	22.3	23.1
650	22.3	23.1
651	22.3	23.1
652	22.3	23.1
653	22.3	23.1
654	22.4	23.2
655	22.4	23.0
656	22.4	23.0
657	22.4	23.0
658	22.3	22.9
659	22.3	22.9
660	22.3	22.9
661	22.3	22.9
662	22.3	22.9
663	22.3	22.9
664	22.2	22.8
665	22.2	22.8
666	22.2	22.8

667	22.2	23.1
668	22.3	23.2
669	22.3	23.2
670	22.3	23.2
671	22.3	23.2
672	22.3	23.2
673	22.3	23.2
674	22.3	23.2
675	22.3	23.2
676	22.3	23.2
677	22.2	23.1
678	22.1	23.0
679	22.2	23.1
680	22.2	23.1
681	22.2	23.1
682	22.2	23.2
683	22.1	23.1
684	22.1	23.1
685	22.1	23.1
686	22.1	23.1
687	22.1	23.1
688	22.0	23.0
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690	22.0	23.0
691	22.0	23.0
692	21.9	22.9
693	21.9	22.9
694	22.0	23.0
695	21.9	22.9
696	22.0	23.0
697	22.0	23.0
698	22.0	23.0
699	22.0	23.0
700	22.0	23.0
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703	22.0	22.9
704	22.0	22.9
705	22.0	22.9
706	22.0	22.9
707	22.1	22.9
708	22.1	22.9

709	22.1	22.9
710	22.1	23.0
711	22.1	23.0
712	22.1	23.0
713	22.1	23.0
714	22.2	23.0
715	22.2	23.0
716	22.1	23.0
717	22.1	23.0
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722	22.2	23.1
723	22.2	23.1
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727	22.3	23.1
728	22.3	23.1
729	22.3	23.1
730	22.2	23.1
731	22.3	23.1
732	22.4	23.1
733	22.4	23.1
734	22.4	23.2
735	22.4	23.2
736	22.3	23.3
737	22.4	23.3
738	22.4	23.3
739	22.3	23.3
740	22.3	23.3
741	22.3	23.3
742	22.3	23.2
743	22.3	23.2
744	22.3	23.2
745	22.3	23.3
746	22.4	23.3
747	22.4	23.3
748	22.4	23.3
749	22.3	23.4
750	22.1	23.2

751	22.2	23.2
752	22.2	23.2
753	22.1	23.2
754	22.1	23.2
755	22.2	23.1
756	22.3	23.1
757	22.2	23.1
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761	22.1	23.0
762	22.1	23.0
763	22.1	23.1
764	22.1	23.0
765	22.1	23.0
766	22.1	23.0
767	22.3	23.0
768	22.3	23.2
769	22.2	23.1
770	22.2	23.2
771	22.1	23.1
772	22.1	23.1
773	22.1	23.0
774	22.0	23.0
775	22.0	23.0
776	22.0	22.9
777	22.0	22.9
778	22.0	22.9
779	22.1	22.9
780	22.1	23.0
781	22.0	23.0
782	22.0	23.0
783	22.0	23.0
784	22.0	23.0
785	22.2	23.0
786	22.1	23.0
787	22.2	23.0
788	22.2	23.1
789	22.1	23.1
790	22.1	23.1
791	22.2	23.1
792	22.1	23.1

793	22.0	23.0
794	22.0	23.0
795	22.0	23.0
796	22.0	23.0
797	22.0	23.0
798	22.0	23.0
799	22.0	23.0
800	22.1	23.0

Test 2:

Time (s)	T (mean, inlet, C)	T (mean, outlet, C)
1	26	26
2	26	26
3	26	26
4	25.4	25.6
5	25	25.3
6	24.3	24.5
7	23.4	23.7
8	23	23.3
9	22.6	23
10	22.4	22.8
11	22.4	22.7
12	22.3	22.7
13	22.3	22.7
14	22.2	22.6
15	22.1	22.6
16	22.1	22.7
17	22.1	22.6
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38	22.1	22.6
39	22.1	22.6

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49	22.2	22.6
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646	22.1	22.8
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666	22.1	22.7
667	22.1	22.7
668	22	22.7
669	22	22.7

670	22	22.7
671	22	22.6
672	22	22.6
673	22	22.6
674	22	22.5
675	22	22.6
676	22	22.6
677	22.1	22.6
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680	22.1	22.5
681	22.1	22.5
682	22.1	22.5
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711	22.1	22.6

712	22.1	22.7
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717	22.1	22.7
718	22.1	22.7
719	22	22.7
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722	22	22.7
723	22	22.7
724	22	22.7
725	22	22.6
726	22	22.6
727	22	22.6
728	22	22.5
729	22	22.5
730	22	22.5
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732	22	22.5
733	22	22.4
734	22	22.4
735	22	22.4
736	22	22.4
737	22	22.4
738	22	22.4
739	22	22.4
740	22.1	22.4
741	22.1	22.4
742	22.1	22.4
743	22.1	22.4
744	22.1	22.5
745	22.1	22.5
746	22.1	22.5
747	22.1	22.5
748	22.1	22.5
749	22.1	22.4
750	22.1	22.4
751	22.1	22.4
752	22.1	22.4
753	22.1	22.5

754	22.1	22.6
755	22.1	22.6
756	22.1	22.6
757	22.1	22.6
758	22.1	22.6
759	22.1	22.6
760	22.1	22.4
761	22.1	22.5
762	22.1	22.4
763	22.1	22.6
764	22.1	22.6
765	22.1	22.7
766	22.1	22.6
767	22.2	22.6
768	22.2	22.6
769	22.2	22.6
770	22.2	22.7
771	22.2	22.7
772	22.2	22.7
773	22.2	22.8
774	22.2	22.8
775	22.1	22.6
776	22.1	22.5
777	22.1	22.5
778	22.1	22.5
779	22.1	22.5
780	22.1	22.4
781	22.1	22.4
782	22.1	22.4
783	22.1	22.4
784	22.1	22.5
785	22.1	22.5
786	22.1	22.4
787	22.1	22.4
788	22.1	22.3
789	22.1	22.5
790	22.1	22.6
791	22.1	22.6
792	22.1	22.6
793	22.1	22.6
794	22.1	22.6
795	22.1	22.6

796	22.1	22.6
797	22.1	22.7
798	22.1	22.7
799	22.1	22.6
800	22.1	22.7

Appendix E: Time Sheet and Costs

Hours:

Research and Design:	30 hours
Construction:	24 hours
Testing:	2.5 hours
Typing:	16 hours

Total: 72.5 hours

Cost:

Building Materials, Home Depot	\$138.78
6061-T6 Aluminum Sheet .020 inch, OnlineMetals.com:	\$84.96
Vernier Lab Cradle, Vernier Technologies:	\$155.00
Vernier Thermocouples, Vernier Technologies:	\$120.00
Pyranometer, Amazon.com:	\$94.14
Infrared Thermometer, Amazon.com:	\$25.00
Solar Engineering of Thermal Processes Textbook, Amazon.com:	\$76.99
Introduction to Heat Transfer Textbook, Cal Poly University Store:	\$163.99
Printing and Binding, Poly Prints:	\$13.19

Total: \$872.05